



# **Parametric Study of Mechanical Behavior of Superconducting Solenoids**

Giacomo Ragni

August 17, 2011



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- 1 Introduction
- 2 Analytical model
- 3 Parametric studies
- 4 Multiple skin configuration
- 5 Future developments



# Introduction

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Superconducting Solenoids in high magnetic fields are heavily loaded.

It is important to calculate the mechanical behavior which depends on a number of parameters.



# Introduction

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Superconducting Solenoids in high magnetic fields are heavily loaded.

It is important to calculate the mechanical behavior which depends on a number of parameters.

A fully analytical model is being realized to perform parametric studies on different configurations of solenoids.

# Accomplished tasks



# Accomplished tasks

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Verify previous analytical model

[E. Terzini and E. Barzi, "Analytical Study of Stress State in HTS Solenoids", FERMILAB-TM-2448-TD]



Understand and run Ansys mesomechanical model

[A. Bartalesi, "Design of High Field Solenoids made of High Temperature Superconductors", FERMILAB-MASTERS-2009-04]

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Improve the accuracy of the analytical model for parametric studies

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Improve the accuracy of the analytical model for parametric studies



Add insert coil configuration to the original self-field analytical model



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Sensitivity analysis of all parameters ( $E, \nu, \mu_0$ )

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Sensitivity analysis of all parameters ( $E$ ,  $\nu$ ,  $\mu_0$ )



Study of reinforced coils with multiple skins



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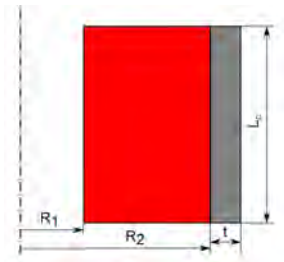
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# Analytical model

## Lamé's equation

$$\frac{E}{1 - \nu^2} \left( \frac{d^2 u}{dr^2} - \frac{1}{r} \frac{du}{dr} - \frac{u}{r^2} \right) + f = 0$$



## Boundary conditions

$$\sigma_{rr,c}(R_1) = 0$$

$$\sigma_{rr,s}(R_2 + t) = 0$$

$$\sigma_{rr,c}(R_2) - \sigma_{rr,s}(R_2) = 0$$

$$u_{c2}(R_2) - u_{s2}(R_2) = 0$$



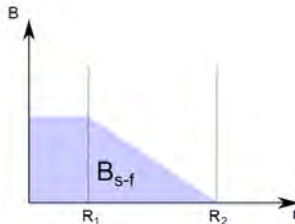
# Magnetic model

## Self-field

$$B_0(\alpha, \beta) = R_1 \mu_0 J \beta \ln \left( \frac{\sqrt{\alpha^2 + \beta^2} + \alpha}{\sqrt{1 + \beta^2} + 1} \right)$$

where  $\alpha = \frac{R_2}{R_1}$  and  $\beta = \frac{L_c}{2} \frac{1}{R_1}$

$$B_0(r) = \begin{cases} B_0 & \text{if } r < R_1 \\ B_0 \left( 1 - \frac{r-R_1}{R_2-R_1} \right) & \text{if } r \geq R_1 \end{cases}$$



$$f = \frac{1 - \nu^2}{E} J(B_0) \frac{B_0}{R_2 - R_1} \left( \frac{R_2 r^2}{3} - \frac{r^3}{8} \right)$$

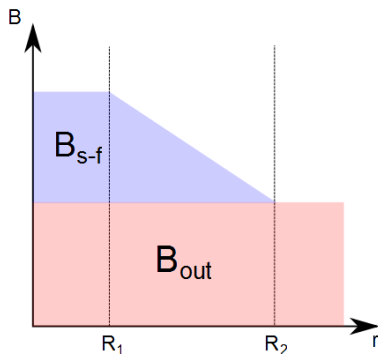


# Magnetic model

## Insert coil

Superposition of self-field and  
outer field

$$B_{out}(r) = \text{const.}$$



$$f = \frac{1 - \nu^2}{E} J(B_{tot}) \left[ \frac{B_0}{R_2 - R_1} \left( \frac{R_2 r^2}{3} - \frac{r^3}{8} \right) + B_{out} \frac{r^2}{3} \right]$$



## Effect of anisotropy



Material	E (GPa)	area (mm <sup>2</sup> )
YBCO	110	0.4
Kapton	5.5	0.1125
Epoxy	4.5	0.05

An averaged Young modulus depending on the ratio among the areas was introduced.

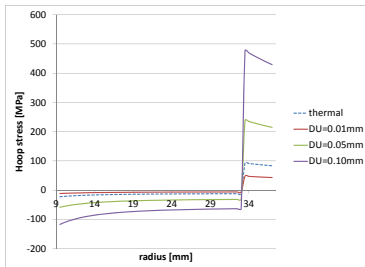
$$E_{av} = 79.7 \text{ GPa}$$



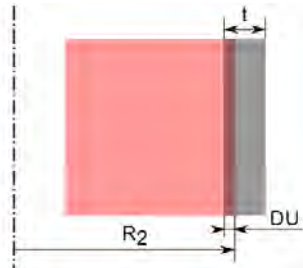
# Temperature $\sim$ Interference

## Thermal deformations

$$\begin{pmatrix} \sigma_{rr} \\ \sigma_{\theta\theta} \end{pmatrix} = \frac{E}{1 - \nu^2} \begin{pmatrix} 1 & \nu \\ \nu & 1 \end{pmatrix} \begin{pmatrix} \epsilon_{rr} - \alpha \Delta T \\ \epsilon_{\theta\theta} - \alpha \Delta T \end{pmatrix}$$



## Assembly interference







# Configuration analyzed

## Geometry

$$R_1 = 9.5mm$$

$$R_2 = 31mm$$

$$L_c = 126mm$$

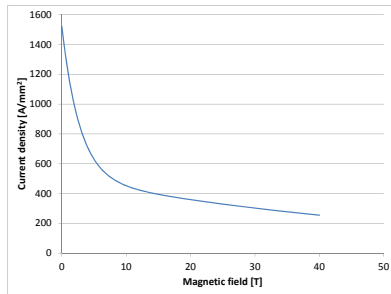
$$t = 4mm$$

## Magnetic properties

$$B = 13.4T$$

$$J(B) = 522.7 \frac{A}{mm^2}$$

## Engineering Current Density





# Configuration analyzed

## Geometry

$$R_1 = 9.5mm$$

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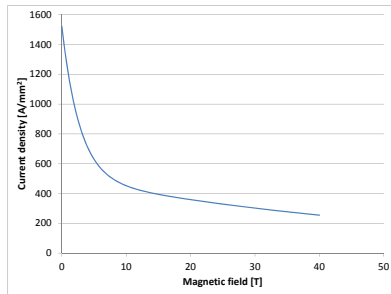
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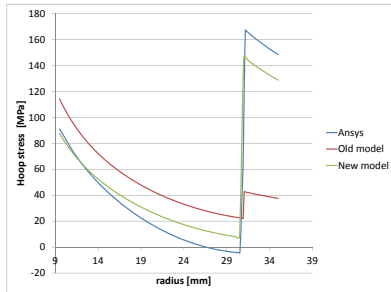


All the stress  
distributions are in the  
mid-plane

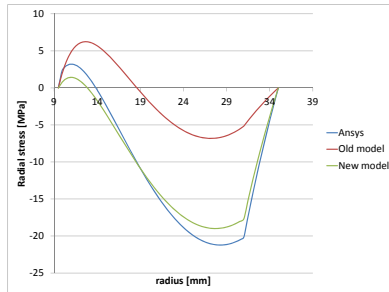


# Results - Self-field

## Hoop stresses



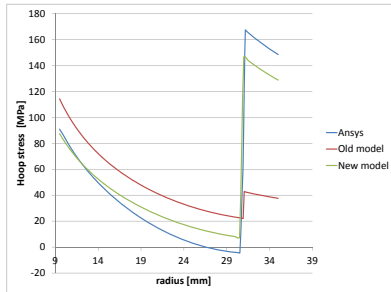
## Radial stresses



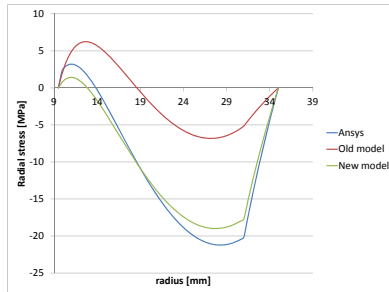


# Results - Self-field

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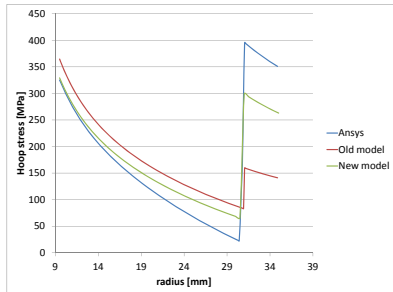
$$e_{max} = 12\%$$



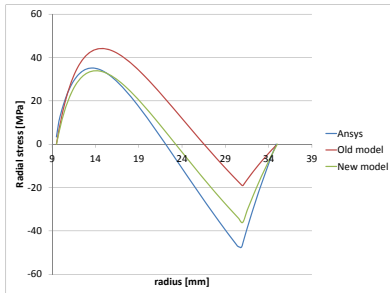
# Results - Insert coil

10 T background field

## Hoop stresses



## Radial stresses

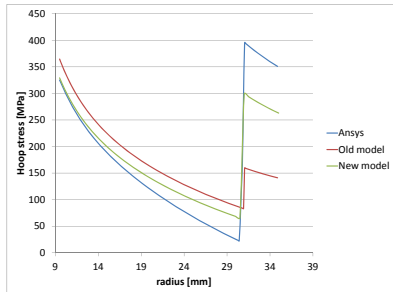




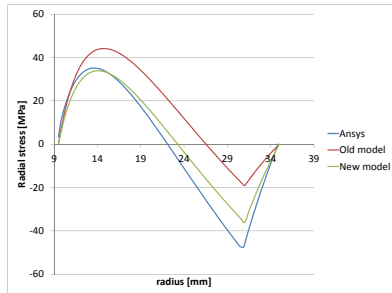
# Results - Insert coil

10 T background field

## Hoop stresses



## Radial stresses



$$e_{max} = 23\%$$



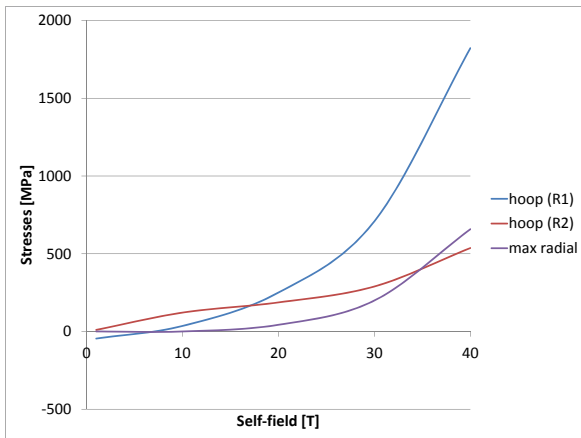
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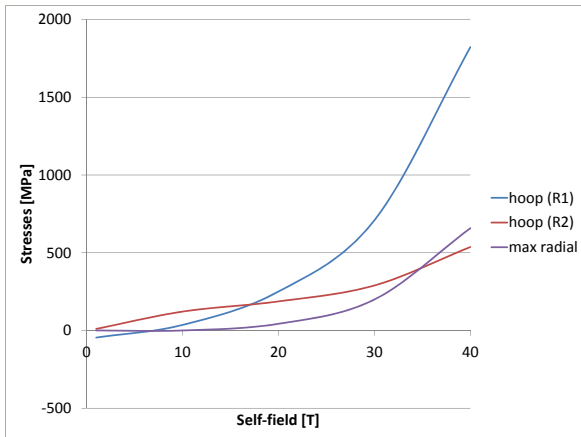
# Max stresses (self-field)







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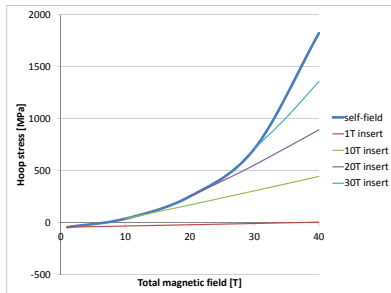


At high fields the coil is the most critical

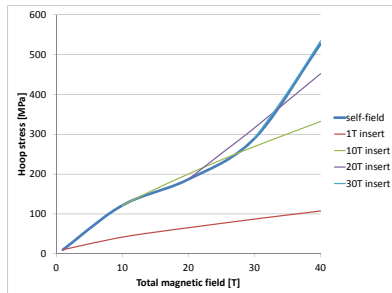


# Max stresses (insert-coil)

## Hoop at $R_1$ (COIL)



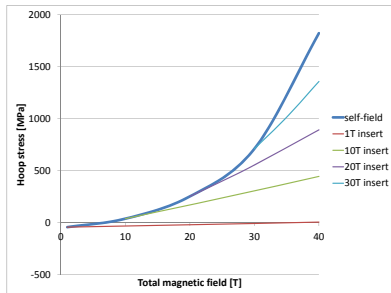
## Hoop at $R_2$ (SKIN)



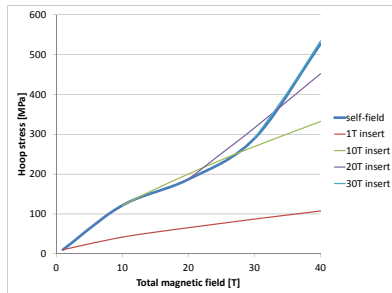


# Max stresses (insert-coil)

## Hoop at $R_1$ (COIL)



## Hoop at $R_2$ (SKIN)



Smaller solenoids have smaller stresses



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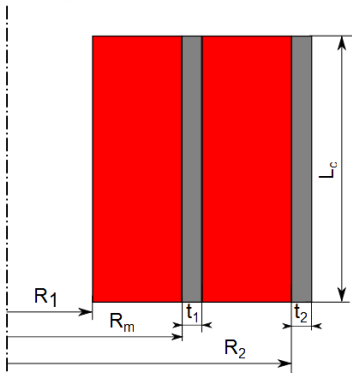
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# Double skin configuration

## Boundary conditions



$$\sigma_{rr,c_1}(R_1) = 0$$

$$\sigma_{rr,s_2}(R_2 + t_2) = 0$$

$$\sigma_{rr,c_1}(R_m) - \sigma_{rr,s_1}(R_m) = 0$$

$$\sigma_{rr,s_1}(R_m + t_1) - \sigma_{rr,c_2}(R_m + t_1) = 0$$

$$\sigma_{rr,c_2}(R_2) - \sigma_{rr,s_2}(R_2) = 0$$

$$u_{c_1}(R_m) - u_{s_1}(R_m) = 0$$

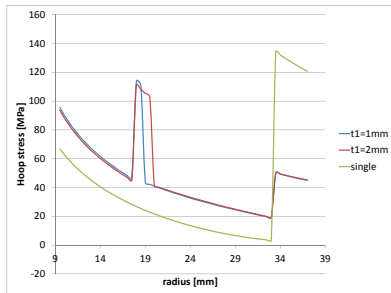
$$u_{s_1}(R_m + t_1) - u_{c_2}(R_m + t_1) = 0$$

$$u_{c_2}(R_2) - u_{s_2}(R_2) = 0$$

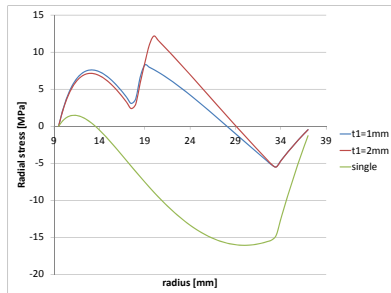


# Effect of steel thickness

## Hoop stress



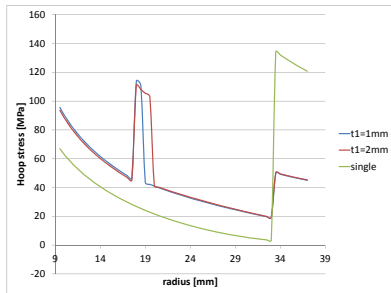
## Radial stress



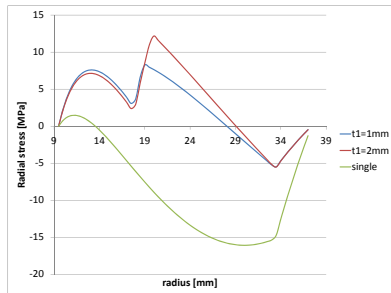


# Effect of steel thickness

## Hoop stress



## Radial stress



Max stresses can be reduced



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# Future developments

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Improve the accuracy of axial stresses



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Improve the accuracy of axial stresses



Optimize geometrical parameters



# Future developments

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Deepen multiple coil configurations [Hahn (MIT)2011]



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Realize technical applications of the analytical model

# Axial stresses

## Ansys model



Glued connection imposed



Over estimation of the stresses

## Analytical model



Impose a constant axial deformation at the mid plane



Consider the effect of the radial component of the magnetic field

$$B_r = \frac{\mu_0 I}{2\pi} \frac{1}{\sqrt{(a+r)^2 + z^2}} \left[ K(k) + \frac{a^2 - r^2 - z^2}{(a-r)^2 + z^2} E(k) \right]$$

